

# Investigation on Structural Stability of CMSX-4 and RENE 77 made Gas Turbine Guide Vanes with Impingement Cooling

Dr. R. Saravanan<sup>1</sup>, M. Karuppasamy<sup>2</sup>

<sup>1</sup>Principal & Professor, Mechanical Engineering, Elenki Institute of Engineering and Technology, Affiliated to JNTU Hyderabad, Patelguda, Hyderabad -502319, TS, India.

<sup>2</sup>Associate Professor, Department of Mechanical Engineering, S.Veerasamy Chettiar College of Engineering and Technology, Puliangudi -627855, Tamil Nadu, India.

**Abstract**— According to the thermodynamics increase of heat input increases the work output as we as efficiency. The gas turbine is a such a thermal device and its inlet temperature is proportional to the thermal efficiency and power output. Higher temperature effects on structural stability of turbine components which expose at elevated temperature. This brings the need for investigation of stability of them before they actually fabricate. This research investigates the structural stability of fixed blade of gas turbine at elevated temperature. The CMSX-4 and RENE 77 are considered for blade materials. The impingement type cooling design is considered for both types of blades. The cooling design is hollow portion and they greatly influence on the structural stability of gas turbine components. This research focuses on the material influence on fixed blade with impingement cooling design. The Pro -E and ANSYS R14 were used for design and FEM analysis respectively.

**Keywords**— Structural stability, ANSYS, Pro-E, Gas turbine, Guide Vane, Impingement.

## I. INTRODUCTION

Applications of gas turbines are wide in range like aircraft, marine propulsion, locomotive, power generation, and other industrial prime movers. For increasing thermal efficiency and work output in gas turbines, the higher input temperatures encountered ranging from 1000 to 1500°C [1-3]. In particularly aerospace applications operates gas turbines higher inlet temperature like 1,727°C [4-6], and with a high pressure ratio about 50 at compressor [6]. Such high temperatures causes thermal damages (melting, corrosion, oxidation and erosion) [7] and degradation of local or global structural strengths of blades, vanes and other components and it was estimated that half of the lifespan of the blades gets reduced due to small temperature difference by improper cooling [3,6,9]. The damages include blade trailing-edge cracks [8], buckling and risk of blade failure [11], thermal-fatigue [8,10,11].

Many studies were conducted on optimization of lip thickness to slot height ratio ( $t/H$ ) in trailing edge cooling of blades and vanes [12 -17] in which Kacker et al. [12,13] considered lip thickness constant to estimate film cooling effectiveness, Taslim et al [14,15] varies slot geometries and blowing ratios. The  $t/H$  ratio from 0.5 to 1, decrease the overall film-cooling effectiveness by about 10% [14-16]. The decreases of  $t/H$  ratio, increases the film-cooling effectiveness [17-19]. [20] considered a a rectangular divergent channel which consists of serpentine shape with ribs, dimples/protrusions, guide vanes, and pin fins at the tip turning the region for his heat transfer studies. [21] studied the cooling performance at tip surfaces of guide vanes and blades at turning regions and insisted the importance of proper design to obtaining desired effects. [22] recommended installing guide vanes in the tip turning regions most suitable way to improve cooling of tip surfaces. [23] insisted that selection appropriate cooling technique with respect to configuration is must. The authors suggested two pass channels cooling at moving blades. This research work investigates with materials behaviors at elevated temperature for impingement cooling design on gas turbine fixed blade (guide vane). The Pro-E and ANSYS are employed to design and analysis. ANSYS is generally the preferred tool for analyzing structural stability. [24] used CATIA and ANSYS to design and investigate the structural stability of various components of Two-Wheeled Inverted Pendulum. In later [25] investigated the suitability of Kevlar29/epoxy composite for drive shaft. The influence of cooling design such as impingement and shower head type on gas turbine guide vanes which are made up of Nimonic 901[26], RENE 77 [27], CMSX 4 [28]. The sample guide vane of gas turbine is shown in Figure I. This research work investigates the material suitability for the gas turbine guide vanes with impingement cooling design. The design softwares like PRO/E and ANSYS 14.5 used for modelling and analysis of material stability under high temperature.

## II. MATERIALS AND METHODS

The materials which considered for analysis are CMSX 4 and RENE 77. The cooling type in guide vane is impingement type. The Structural Stability investigations are by means of Displacement analysis, Stress analysis and strain analysis. The Design softwares involved are Pro-E for guide vane design, modelling and ANSYS 14.5 for performing Finite Element Analysis.

### *CMSX 4*

It is a single crystal super alloy and gives more stability at elevated temperature. Its significant properties at high temperatures includes impact strength, rupture strength, fracture toughness. The blades which made up of CMSX-4, exhibited good performances as well as extended life span [29-32]. Hence its physical and mechanical properties were included in this analysis of structural stability of specified cooling designed guide vanes of gas turbine and reduced weight [30].

### *RENE 77*

It is preferred for high temperature applications about 1000°C like aviation, petroleum, gas turbine, space flight, ship, etc. because of its significant properties at elevated temperatures like excellent oxidation resistance, with stand at long term stress, creep properties, reliable in physical and chemical properties, good impact and toughness strengths etc. values of such unique properties were included in the analysis.

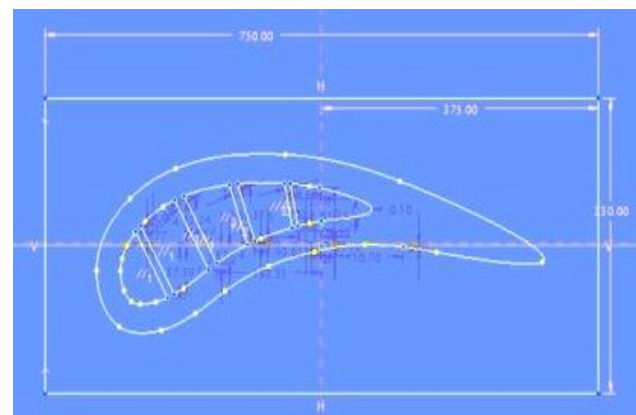
### *Structural Analysis*

The gas turbine guide vane is modelled in Pro-E. Then it is used for meshing and conducting structural analysis in ANSYS 14.5 work bench. The dimensional particulars of the guide vane with impingement cooling are furnished in Figure II. The 3D meshed model is presented in Figure III. The structural analysis like stress analysis, strain analysis and displacement analysis were considered. The comparative study of materials involved in manufacture them is focused on this research. The young's modulus 294000 MPa, material density 0.0000774 kg/mm<sup>3</sup>, Poisson Ratio 0.3 were considered for CMSX 4 made blades. In case of RENE 77 made blades, the young's modulus 200000 MPa, Poisson Ratio 0.30, material density 0.000077 kg/mm<sup>3</sup>. In the meshed model made with 186 nodes, and pressure 0.188 N/mm<sup>2</sup>. The displacement analysis carried at ANSYS 14.5 work bench.

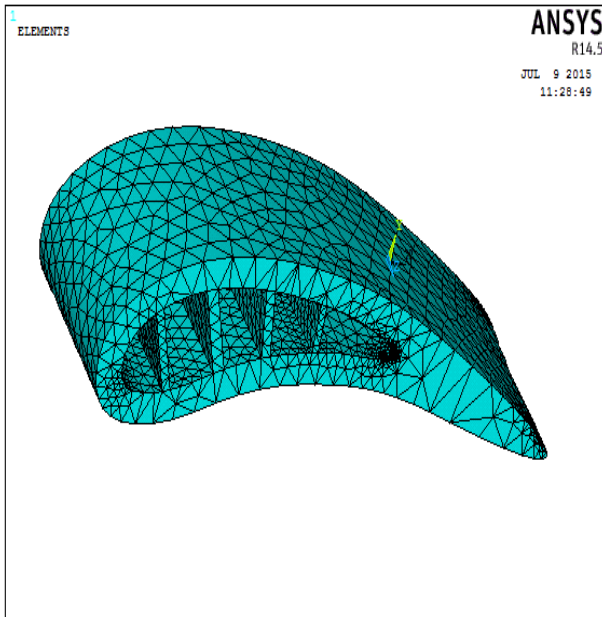
The displacement with respect to turbine load was observed and shown them for CMSX 4 made blades in Figure IV and for RENE 77 made blades in Figure V. The maximum displacements were observed on the concave side of cooling passage nearby its sharp turning. The stress analysis for the above said blades (Figure VI for CMSX 4 made blades and Figure VII for RENE 77 made blades). The strain analysis details are shown in Figure VIII for CMSX 4 made blades and Figure 9 for RENE 77 made blades.



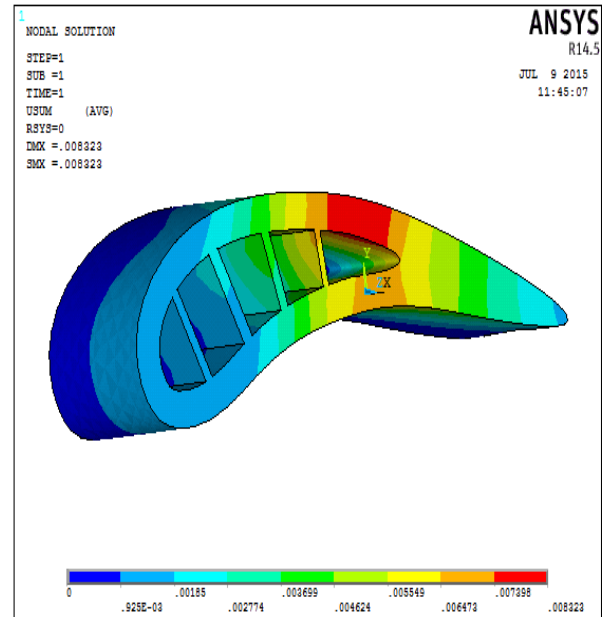
**Figure I Fixed blade (Guide vane)**



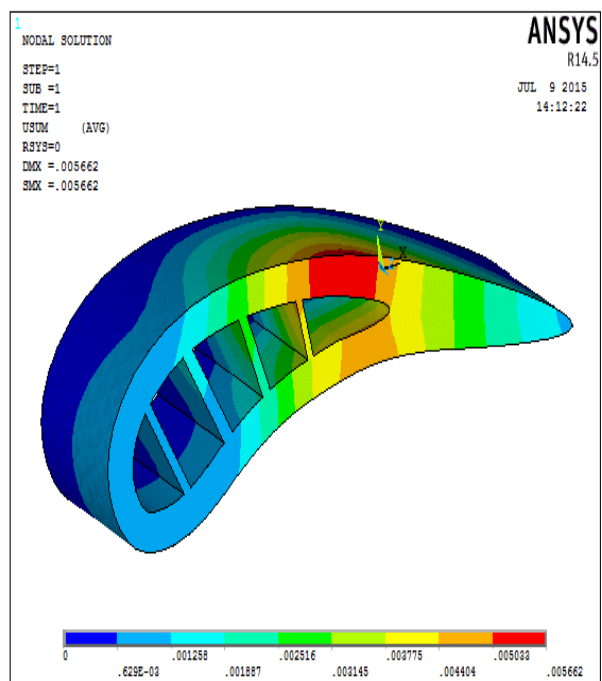
**Figure II 2D-model of Guide vane with impingement cooling design**



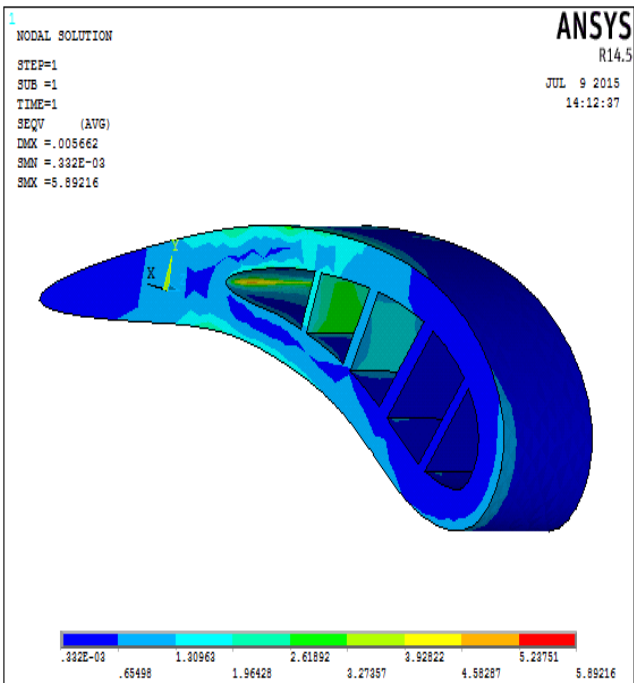
**Figure III** Meshed model of Guide vane with impingement cooling design



**Figure V** The displacement analysis for RENE 77 made blades



**Figure IV** The displacement analysis for CMSX 4 made blades



**Figure VI** The stress analysis for CMSX 4 made blades

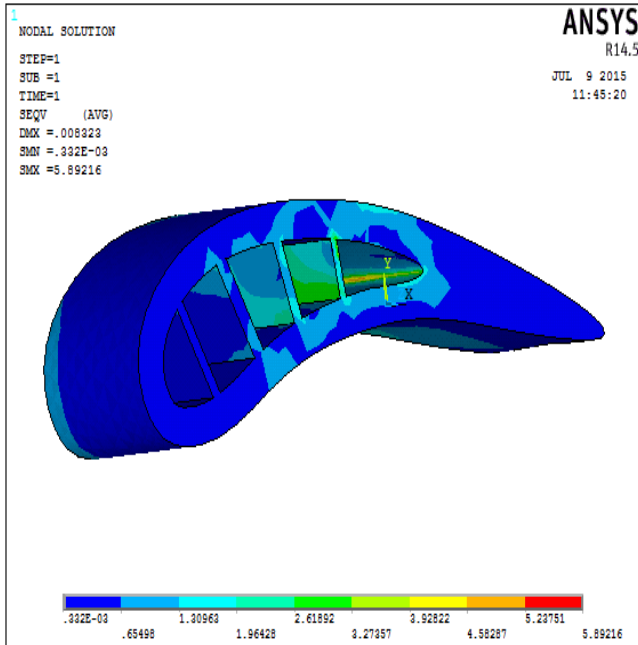


Figure VII The stress analysis for RENE 77 made blades

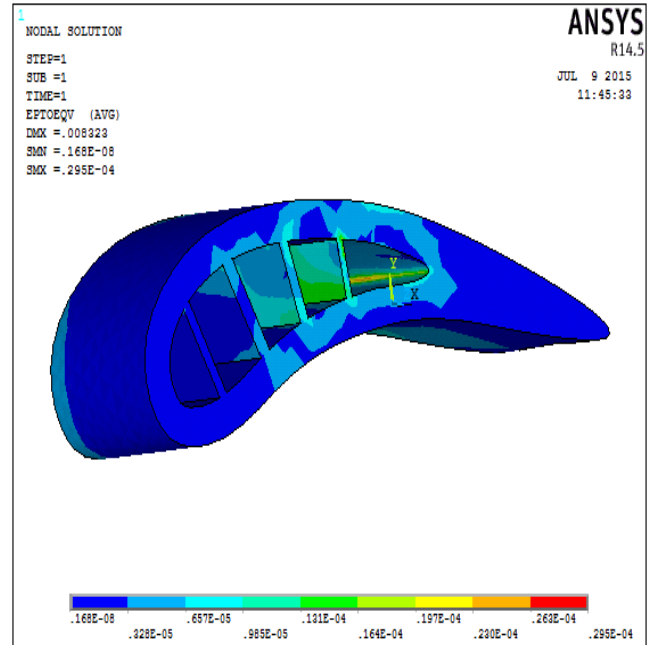


Figure IX The strain analysis for RENE 77 made blades

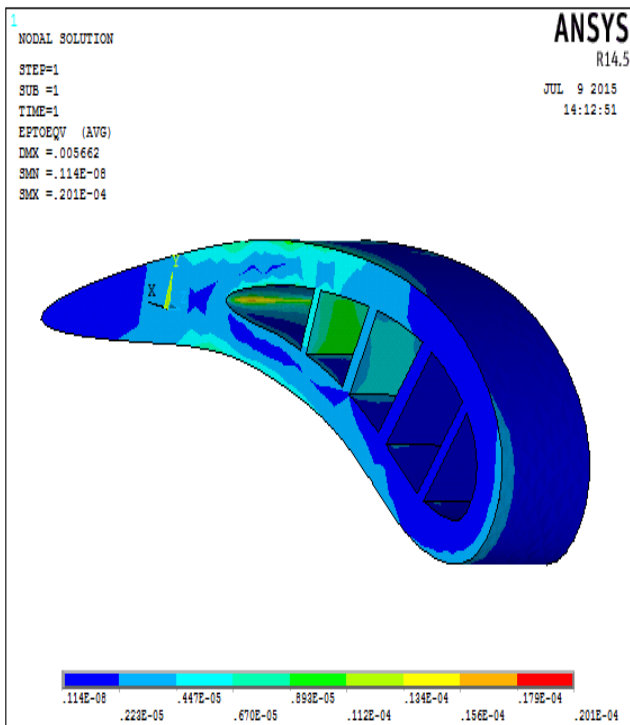


Figure VIII The strain analysis for CMSX 4 made blades

**TABLE I**  
**RESULTS OF STRUCTURAL ANALYSIS**

Guide Vane Made Up of	Displacement (mm)	Stress (N/mm <sup>2</sup> )	Strain
CMSX-4	0.005662	5.89216	0.201e-4
RENE-77	0.008323	5.89216	0.295e-4

### III. RESULTS AND DISCUSSIONS

The structural analysis on CMSX 4 and RENE 77 made guide vanes with impingement type cooling design were carried out and reported results in the Table I. The results are compared with respect to materials for displacement in Figure X, for stress in Figure XII and for strain is in Figure 12. From the results, it is revealed that both the materials are performing well under in the tested condition. The stress bearing capacity per square millimeter little higher for CMSX 4 (Refer Figure X to Figure XII). The RENE 77 little more flexible. The higher values of stress, strain and displacements were noticed at and nearby sharp turning of cooling design.

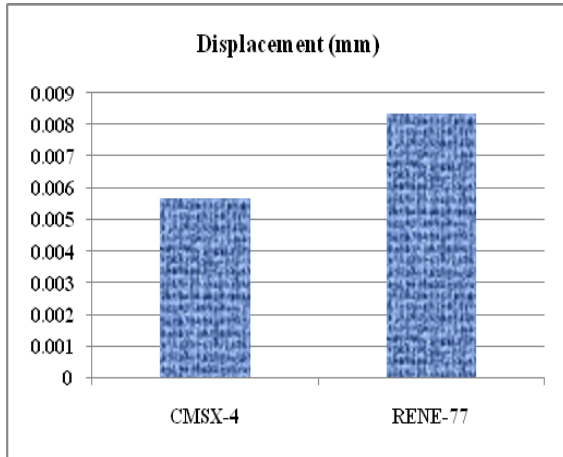


Figure X. The Results of Displacement Analysis

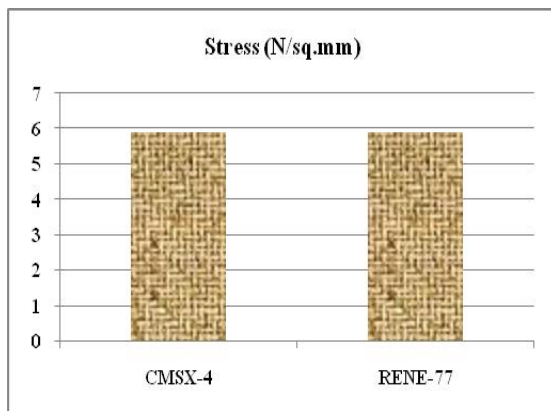


Figure XI. The Results of Stress Analysis

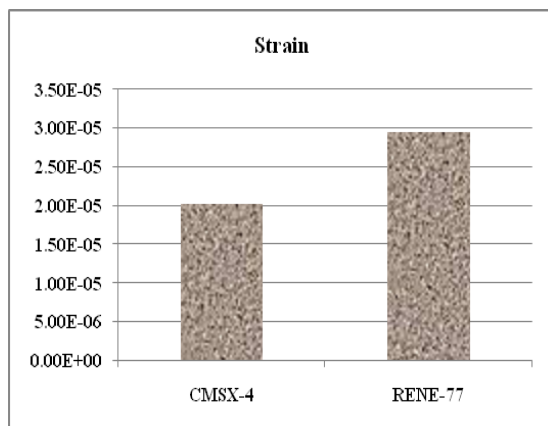


Figure XII. The Results of Strain Analysis

#### IV. RESULTS AND DISCUSSIONS

The structural analysis on CMSX 4 and RENE 77 made guide vanes with impingement type cooling design were carried out and reported results in the Table1. The results are compared in terms of displacement in Figure X, stress in Figure XI and for strain is in Figure XII. From the results, it is revealed that both the materials are performing well under in the tested condition. The stress bearing capacity is a little higher for CMSX 4 made blades than RENE77 made blades (Refer Figure X to Figure XII). The RENE 77 little more flexible. The higher values of stress, strain and displacements were noticed at and nearby sharp turning of cooling design.

#### V. CONCLUSION

The fixed blade of Gas turbine with impingement cooling design considered with two different materials (CMSX 4 and RENE 77). The both materials are capable of withstanding loads at elevated temperature. In which the RENE 77 is little more flexible than CMSX 4. The higher values of stress, strain and displacements were observed at the narrow turning of cooling design. The narrow turning in the cooling design may be designed as wide as much as possible to avoid stress concentration.

#### REFERENCES

- [1] J. C. Han and S. Ekkad, 2001. Recent Studies in Turbine Blade Film Cooling”, J. Rotating Machinery. vol.7, no.1, pp.21–40.
- [2] J. C. Han, 2004. Recent Studies in Turbine Blade Cooling, J. Rotating Machinery, vol. 10, no. 6, pp. 443–457.
- [3] Facchini B, Innocenti L, Tarchi L. 2004. Pedestal and Endwall Contribution in Heat Transfer in Thin Wedge Shaped Trailing Edge. ASME. Turbo Expo: Power for Land, Sea, and Air, vol. 3, pp.101-111.
- [4] Martini P, Schulz A, Wittig S. 2003. Experimental and Numerical Investigation of Trailing Edge Film Cooling by Circular Coolant Wall Jets Ejected From a Slot with Internal Rib Arrays. ASME. Turbo Expo: Power for Land, Sea, and Air, Parts A and B, vol.5, pp.71-79.
- [5] P. Martini and A. Schulz, 2004. Experimental and Numerical Investigation of Trailing Edge Film Cooling by Circular Wall Jets Ejected from a Slot with Internal Rib Arrays, J. Turbo machinery, vol.126, no.2, pp.229– 236.
- [6] Sunden B, Xie G. 2010. Gas turbine blade tip heat transfer and cooling: a literature survey. Heat Transf Engineering, vol. 31, pp.527-54.
- [7] T. Horbach, A. Schulz and H. -J. Bauer, 2011. Trailing Edge Film Cooling of Gas Turbine Airfoils – External Cooling Performance of Various Internal Pin Fin Configurations, J. Turbomachinery, vol. 133, no. 94, pp.041006-1 – 041006-9.
- [8] Z. Yang and H. Hu, 2012. An Experimental Investigation on the Trailing-edge Cooling of Turbine Blades, J. Propulsion and Power Research, vol.1, no.1, pp. 36–47.

- [9] Facchini B, Simonetti F, Tarchi L, 2009. Experimental Investigation of Turning Flow Effects on Innovative Trailing Edge Cooling Configurations With Enlarged Pedestals and Square or Semicircular Ribs, ASME. Turbo Expo: Power for Land, Sea, and Air, Heat Transfer, Parts A and B: pp.795-806.
- [10] J. Choi, S. Mhetras, J. -C. Han, S. Lau and R. Rudolph, 2008. Film Cooling and Heat Transfer on Two Cutback Trailing Edge Model with Internal Performance Blockage, J. Heat Transfer, vol.130, no.1, pp.012201-012213.
- [11] L. Brundage, M. W. Plesniak, P. B. Lawless and S. Ramadhyani, 2007. Experimental Investigation of Airfoil Trailing Edge Heat Transfer and Aerodynamic Losses, J. Experimental Thermal and Fluid Science, vol.31, no.3, pp.249-260.
- [12] S. C. Kacker and J. Whitelaw, 1968. The Effect of Slot Height and Slot-Turbulence Intensity on the Effectiveness of the Uniform Density, Two Dimensional Wall Jet. J. Heat Transfer, vol.90, no.4, pp.469-475.
- [13] S. C. Kacker and J. H. Whitelaw, 1969. An Experimental Investigation of Slot Lip Thickness on Impervious Wall Effectiveness of the Uniform Density, Two-Dimensional Wall Jet, J. Heat and Mass Transfer, vol. 12, no.9, pp. 1196-1201.
- [14] N. E. Taslim, S. D. Spring and B. P. Mehlmann, 1990. An Experimental Investigation of Film Cooling Effectiveness for Slot Various Exit Geometries, Journal of Thermophysics and Heat Transfer, vol.6, no.2, pp.302-307.
- [15] N. E. Taslim, S. D. Spring and B. P. Mehlmann, (1992), Experimental Investigation of Film Cooling Effectiveness for Slot of Various Exit Geometries, J. Thermophysics and Heat Transfer, vol. 6, no.2, pp.302-307.
- [16] F. J. Cunha and M. K. Chyu, 2006. Trailing-Edge Cooling for Gas Turbines, J. Propulsion and Power, vol.22, no.2, pp.286-300.
- [17] R. J. Goldstein, 1971. Film Cooling, J. Advance Heat Transfer, vol.7, pp.321-379.
- [18] S. Sivasegaram and J. Whitelaw, 1969. Film Cooling Slots: The Importance of Lip Thickness and Injection Angle, J. Mechanical Engineering Science, vol.11, no.1, pp. 22-27.
- [19] W. Burns and J. Stollery, 1969. The Influence of Foreign Gas Injection and Slot Geometry on Film Cooling Effectiveness, J. Heat and Mass Transfer, vol.12, no.8, pp. 935-951.
- [20] Lee MS, Jeong SS, Ahn SW, Han JC, 2014. Effects of angled ribs on turbulent heat transfer and friction factors in a rectangular divergent channel, Int J Therm Sci., vol.84, pp.1-8.
- [21] Xie Gongnan, Zhang Weihong, Sund\_en Bengt, 2012. Computational analysis of the influences of guide ribs/vanes on enhanced heat transfer of a turbine blade tip-wall, Int J Therm Sci., vol.51, pp.184-194.
- [22] Lei Jiang, Li SJ, Han JC, Zhang L, Moon HK, 2014. Effect of a turning vane on heat transfer in rotating multipass rectangular smooth channel, J Thermophys Heat Transf., vol.28, no.3, pp.417-427.
- [23] Wang Chenglong, Wang Lei, Sund\_en Bengt, Heat transfer and pressure drop in a smooth and ribbed turn region of a two-pass channel, Appl Therm Eng., vol.85, pp.225-233.
- [24] R. Saravanan, R. Pugazhenthii, P. Vivek and M. Santhanam, 2015. Design and Simulation of a Two-Wheeled Inverted Pendulum - a Balanced, Easy Moving Vehicle for the Material Handling, American-Eurasian Journal of Scientific Research, vol.11, no.3, pp.189-198, 2016.
- [25] R. Saravanan, P Vivek, T Vinod Kumar, 2016. Is Kevlar29/Epoxy Composite an Alternate for Drive Shaft?, Journal of Advances in Mechanical Engineering and Science, vol.2, no.3, pp.1-13.
- [26] Dr.R.Saravanan and G.Vinoth Reddy, 2017, Structural Investigation on Nemoni-901 Made Gas Turbine Guide Vanes, Global Journal of Engineering Science and Research Management, vol.4, no.4, pp-89-94.
- [27] Dr.R.Saravanan and G.Vinoth Reddy, 2017, Structural Investigation on Cooling Design influences in Rene-77 made Gas Turbine Guide Vanes, International Journal of Advanced Scientific Technologies, Engineering and Management Sciences, vol.3, no.5, pp.7-11.
- [28] Dr.R.Saravanan and G.Vinoth Reddy, 2017, Investigation On Influence of Cooling Design In Structural Stability Of CMSX-4 Made Gas Turbine Guide Vanes, International Journal of Science and Research, vol. 6, no. 4., pp. 2522-2526.
- [29] J.Lapin, T.Pelachová, M.Gebura, 2012. The effect of creep exposure on microstructure stability and tensile properties of single crystal nickel based superalloy CMSX-4 Kovove Materials, 50(6), pp.379-386,
- [30] A.Ma, D. Dye, R.C. Reed, 2008. A model for the creep deformation behavior of single-crystal superalloy CMSX-4, Acta Mater., vol.56, no.8, pp.1657-1670.
- [31] J.Lapin, M.Gebura, T.Pelachová, M.Nazmy, 2008. Coarsening kinetics of cuboidal gamma prime precipitates in single crystal nickel base superalloy CMSX-4, Kovove Materials, vol.46, no.6, pp.313-322,
- [32] Juraj Lapin, Tatiana Pelachová, Oto Bajana, 2013. The Effect of Microstructure on Mechanical Properties of Single Crystal CMSX-4 Superalloy, Proceedings of Metal 2013, Brno, Czech Republic, EU., vol. 5, pp.15-17,